

# Should You Consider the Impact of Moisture Storage in Building Materials?

ASHRAE Seminar 54 - Energy Efficient Design of Dedicated Outdoor Air Systems

January 31, 2007

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# What is Moisture Storage?

*or Moisture Capacitance*

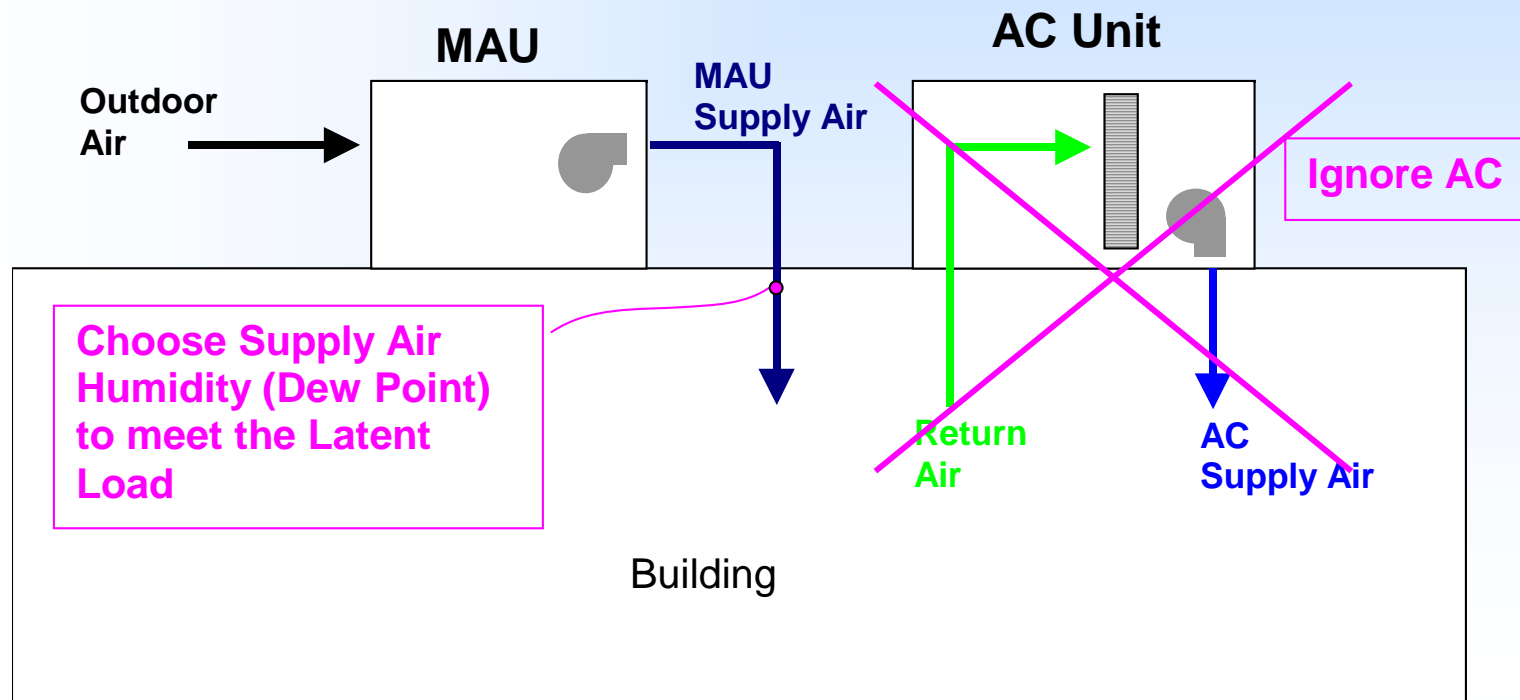
- Zone Air, Walls and Furnishings can hold moisture
  - Building materials adsorb moisture from the zone air
  - Moisture capacitance slows down changes in zone humidity
  - Analogous to thermal capacitance for cooling
- How does this Affect MAU Design?
  - Steady-state moisture load calculations tend to overpredict MAU size requirements
  - High Dew Pt conditions only last a few hours

# MAU Design

*(or Dedicated Outside Air System)*

## Design Approach:

- Precondition ventilation air separately
- Select supply air dew pt to maintain desired space humidity at “worst” conditions



# Traditional Moisture Load Calculations

Moisture gains = Dehumidification Capacity

- Moisture gains: People & Infiltration
- DH Capacity: Dry supply air &  
AC capacity

Load Calculation Procedure

1. Specify the internal loads, ventilation rate and space humidity set point
2. Calculate required supply air humidity

# Traditional Mass Balance

Moisture Gains

Dehumidification Capacity

$$\underbrace{Q_{people} + \dot{m}_{inf} (w_{outdoor} - w)}_{\text{Moisture Gains}} = \underbrace{Q_{AC} + \dot{m}_{vent} (w - w_{supply})}_{\text{Dehumidification Capacity}}$$

# With Moisture Capacitance

Moisture Gains

Dehumidification Capacity

$$C_a \frac{dw}{dt} = \underbrace{\left[ Q_{people} + \dot{m}_{inf} (w_{outdoor} - w) \right]}_{\text{Moisture Gains}} - \underbrace{\left[ Q_{AC} + \dot{m}_{vent} (w - w_{supply}) \right]}_{\text{Dehumidification Capacity}}$$

$w$  = humidity ratio (lb/lb)

$Q_{people}$  = generation rate (lb/h)

$m$  = air flow rate (lb/h)

$Q_{AC}$  = dehumidification rate (lb/h)

# What Should Capacitance Factor Be?

$$C_a = [\rho \cdot V] \times N$$

- N is a multiplier times zone air mass
- Shirey at FSEC has used detailed building simulations to estimate the multiplier N
- Moisture storage is actually decoupled from air mass or node (phase lag)
- Typically **N=10-20** for most building applications

# Steady-State Calculations

$$w = \frac{(Q_{people} + \dot{m}_{inf} w_{outdoor} - Q_{AC} + \dot{m}_{vent} w_{supply})}{(\dot{m}_{inf} + \dot{m}_{vent})}$$

# Transient Calculations

$$w^{j+1} = w^j + \frac{\Delta t}{C_a} [Q_{people} + \dot{m}_{inf} (w_{outdoor}^j - w^j) - \dot{m}_{vent} (w^j - w_{supply})]$$

$w$  = humidity ratio (lb/lb)

$Q_{people}$  = generation rate (lb/h)

$\dot{m}$  = air flow rate (lb/h)

$Q_{AC}$  = dehumidification rate (lb/h)

# Jacksonville

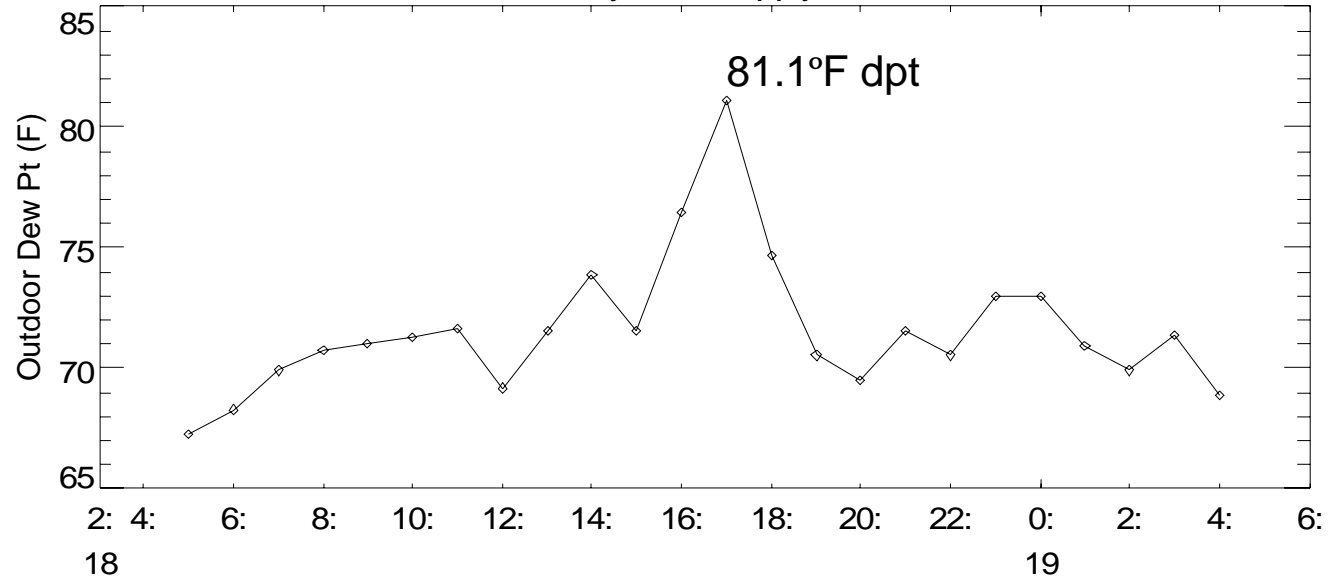
## Classroom:

1,500 sq ft, 0.1 ACH

30 people 15 cfm/p

MAU: 450 cfm  
52°F dpt

JCKSNVLL - July 18 - Supply Air TDP = 52F

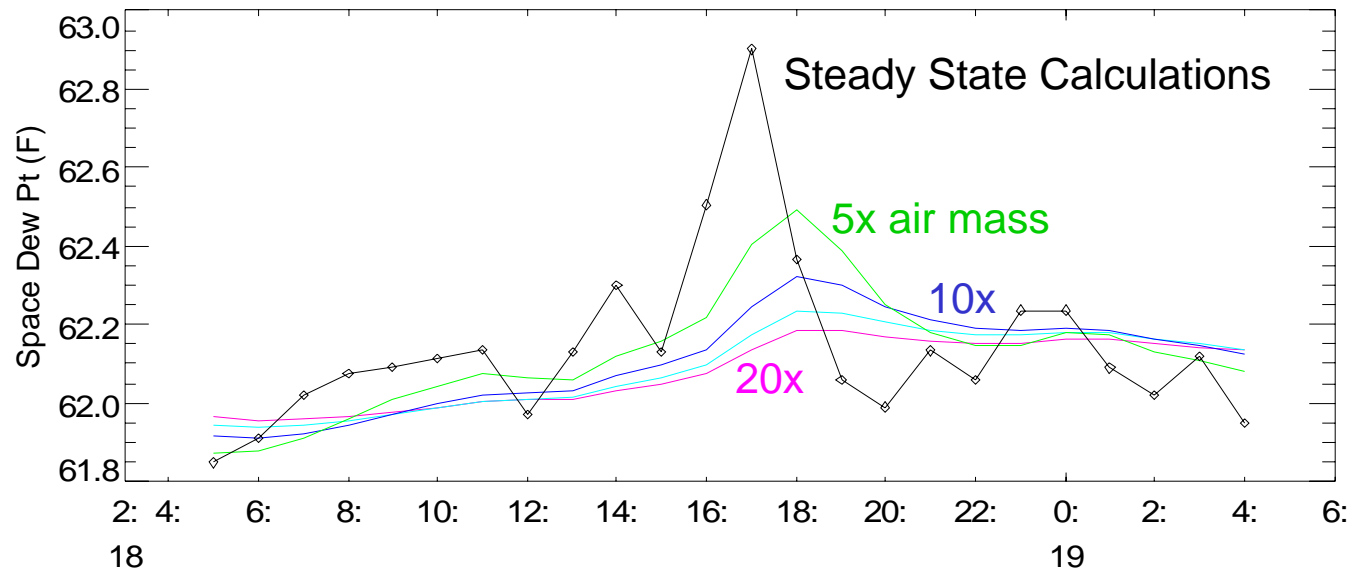


## Space Dew Pt:

62.9°F – SS calcs

62.2°F – 20x air mass

0.7°F – “error”





# Miami

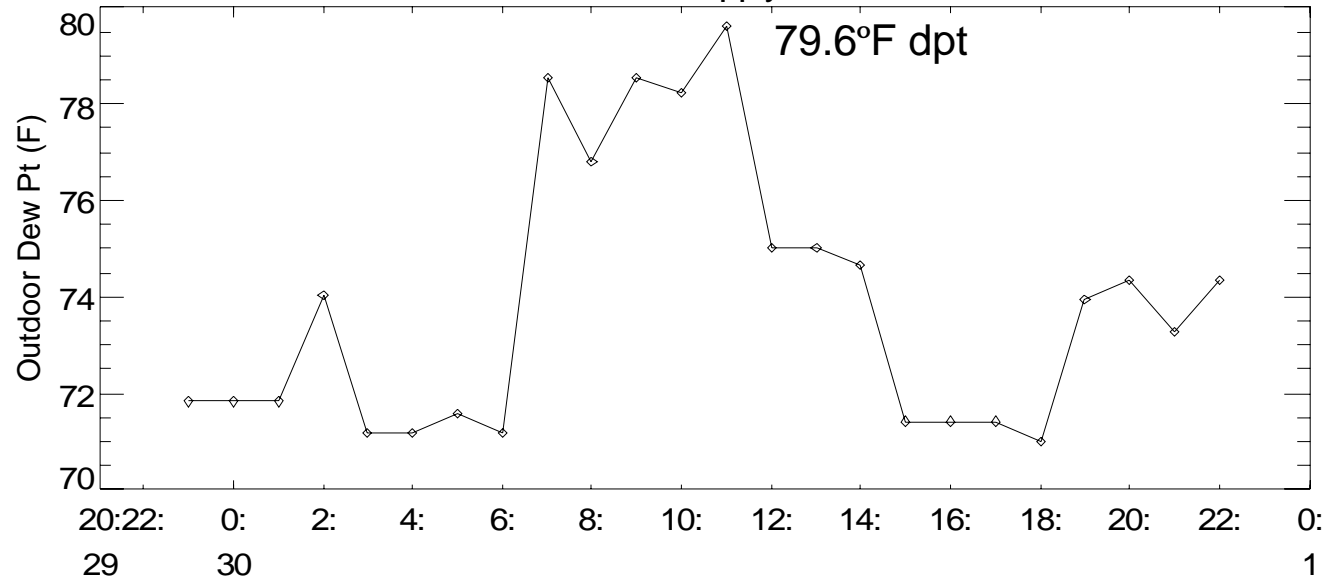
## Classroom:

1,500 sq ft, 0.1 ACH

30 people 15 cfm/p

MAU: 450 cfm  
52°F dpt

miami - June 30 - Supply Air TDP = 52F

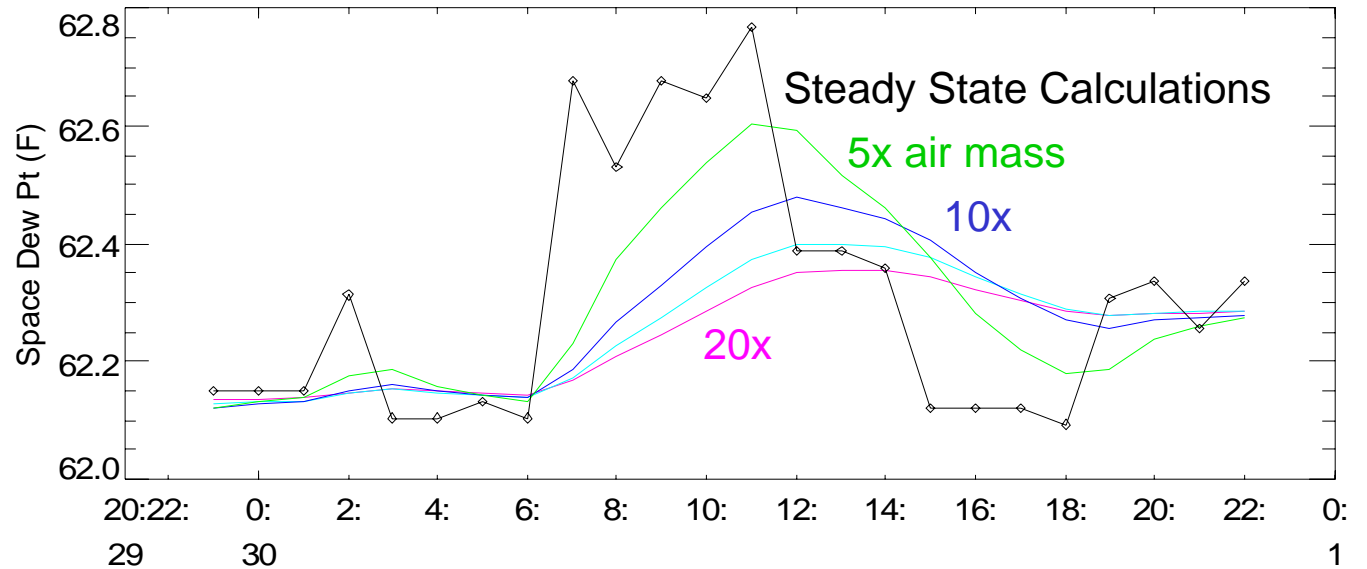


## Space Dew Pt:

62.8°F – SS calcs

62.4°F – 20x air mass

0.4°F – “error”



# Houston

## Classroom:

1,500 sq ft, 0.1 ACH

30 people 15 cfm/p

MAU: 450 cfm  
52°F dpt

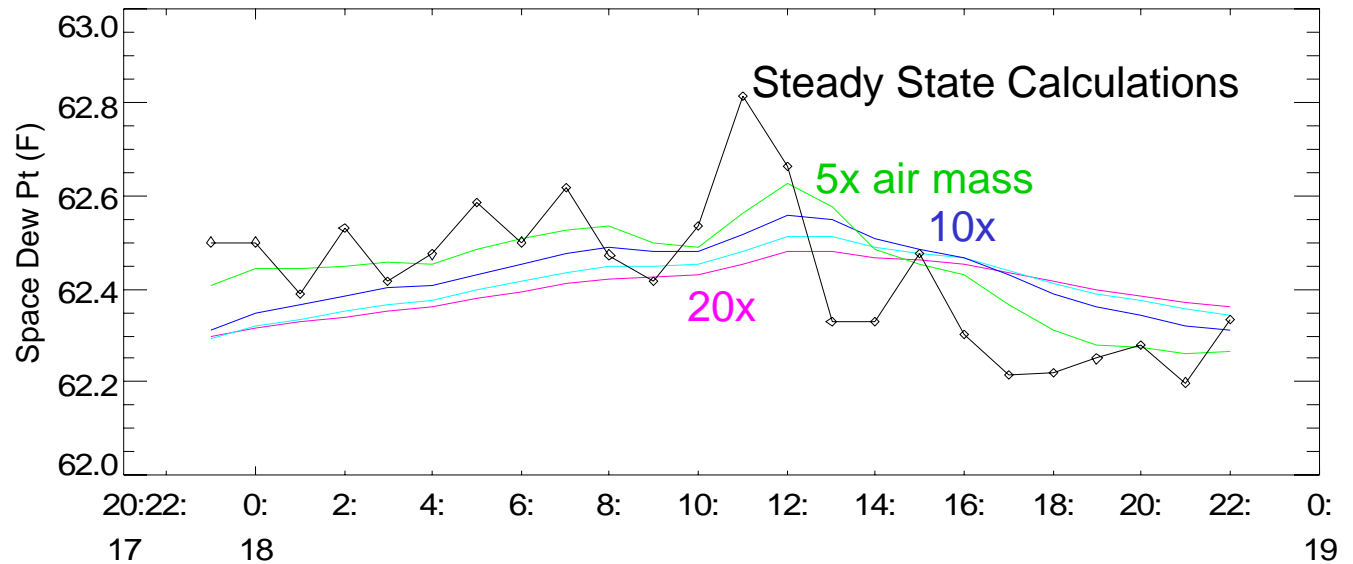
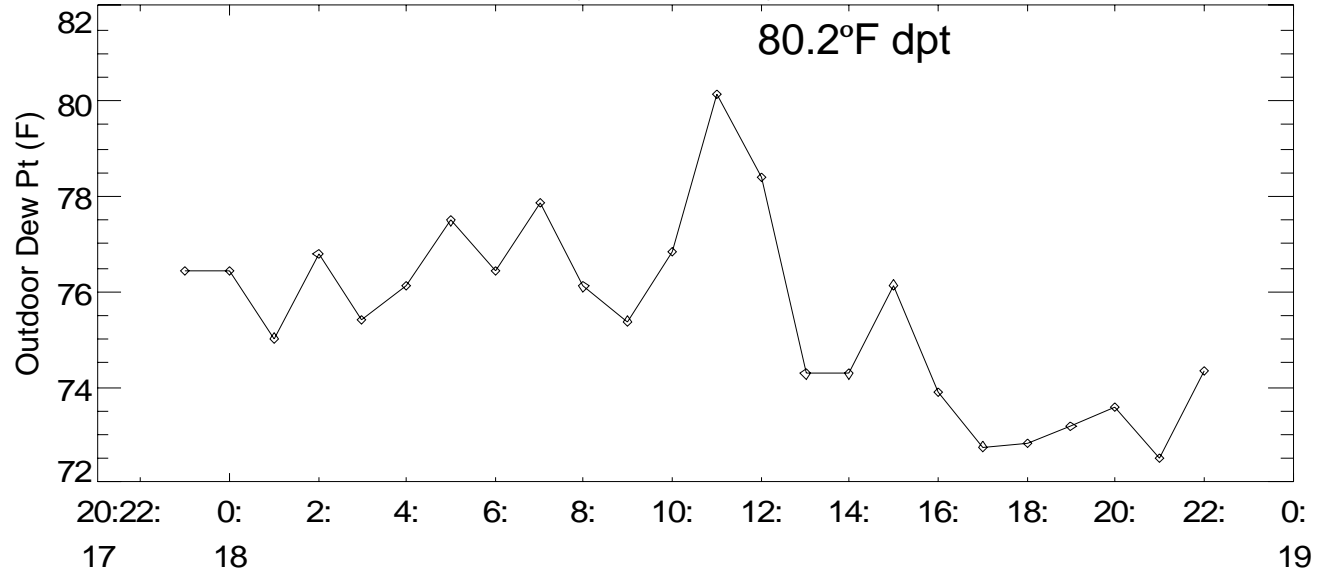
## Space Dew Pt:

62.8°F – SS calcs

62.5°F – 20x air mass

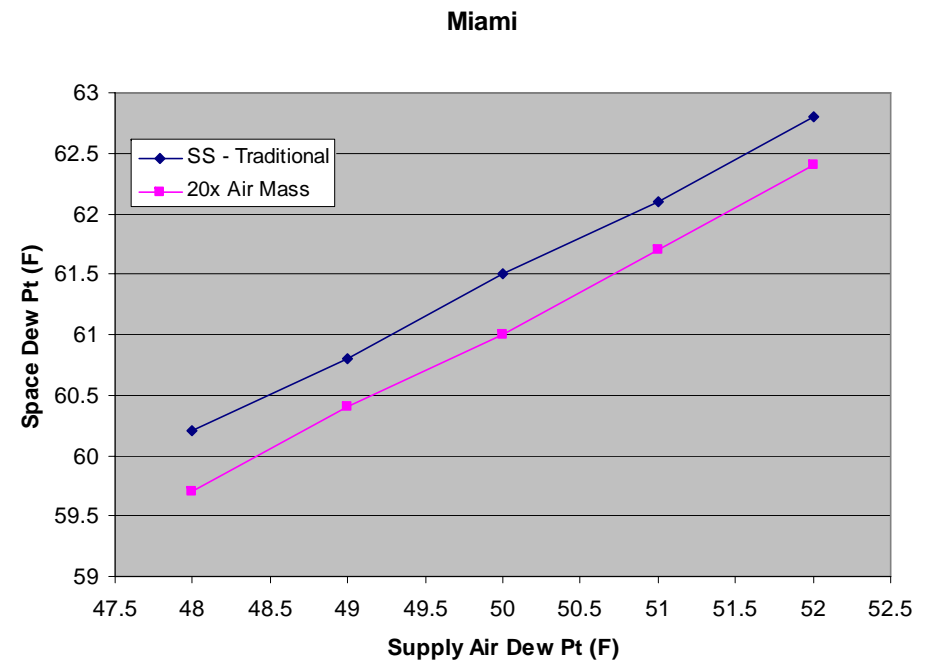
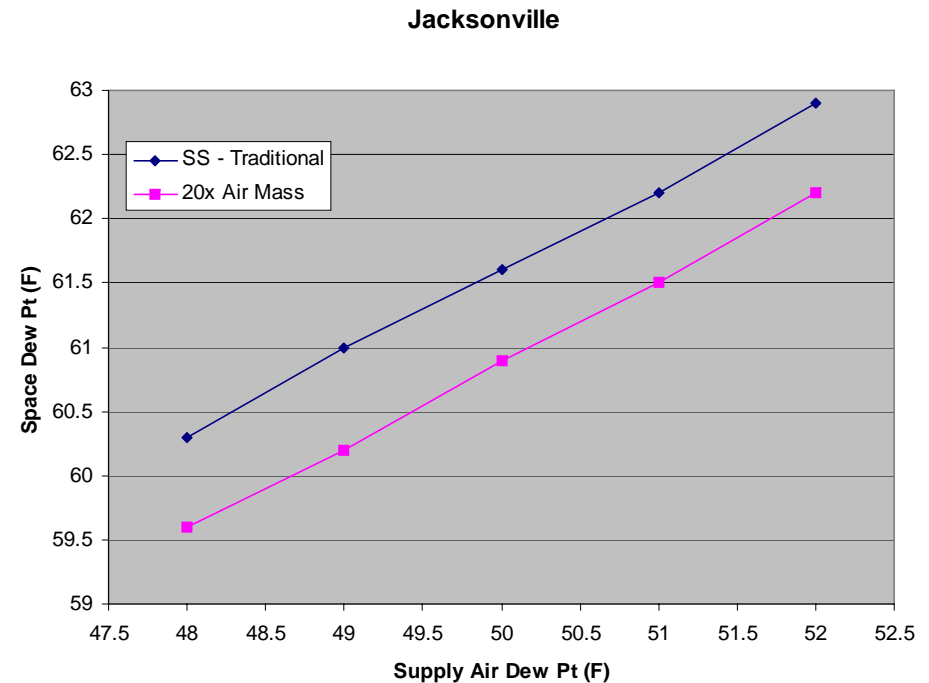
0.3°F – “error”

Houston - July 18 - Supply Air TDP = 52F



# Impact of Moisture Capacitance

- Predicts lower space dew point by 0.3-0.7°F
- About the same impact as increasing supply air dew point by 1°F
- But even more impact in other building applications!



# Jacksonville

## Office:

1,500 sq ft, 0.1 ACH

**10 people** 15 cfm/p

MAU: **150 cfm**  
52°F dpt

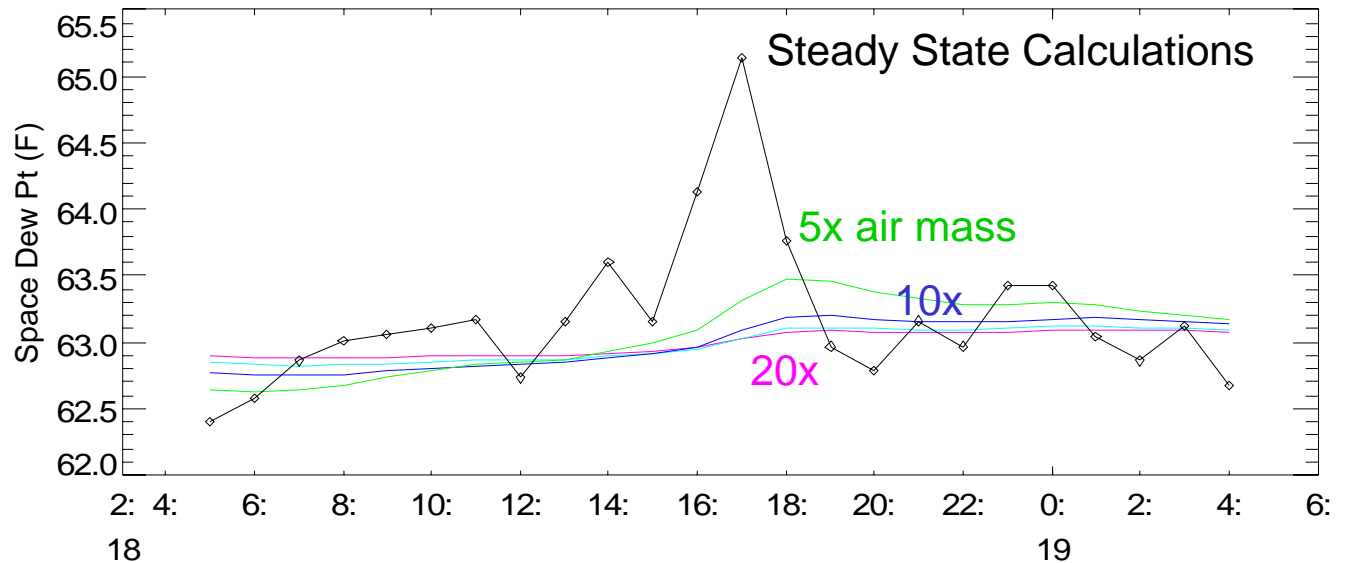
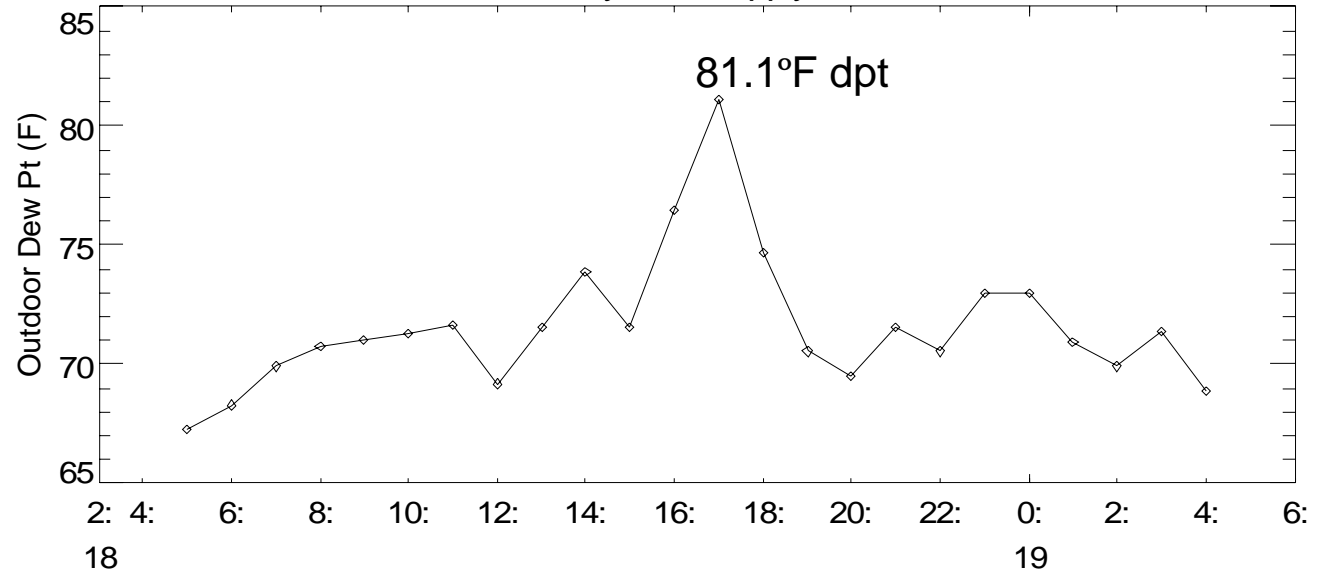
## Space Dew Pt:

65.1°F – SS calcs

**63.1°F – 20x air mass**

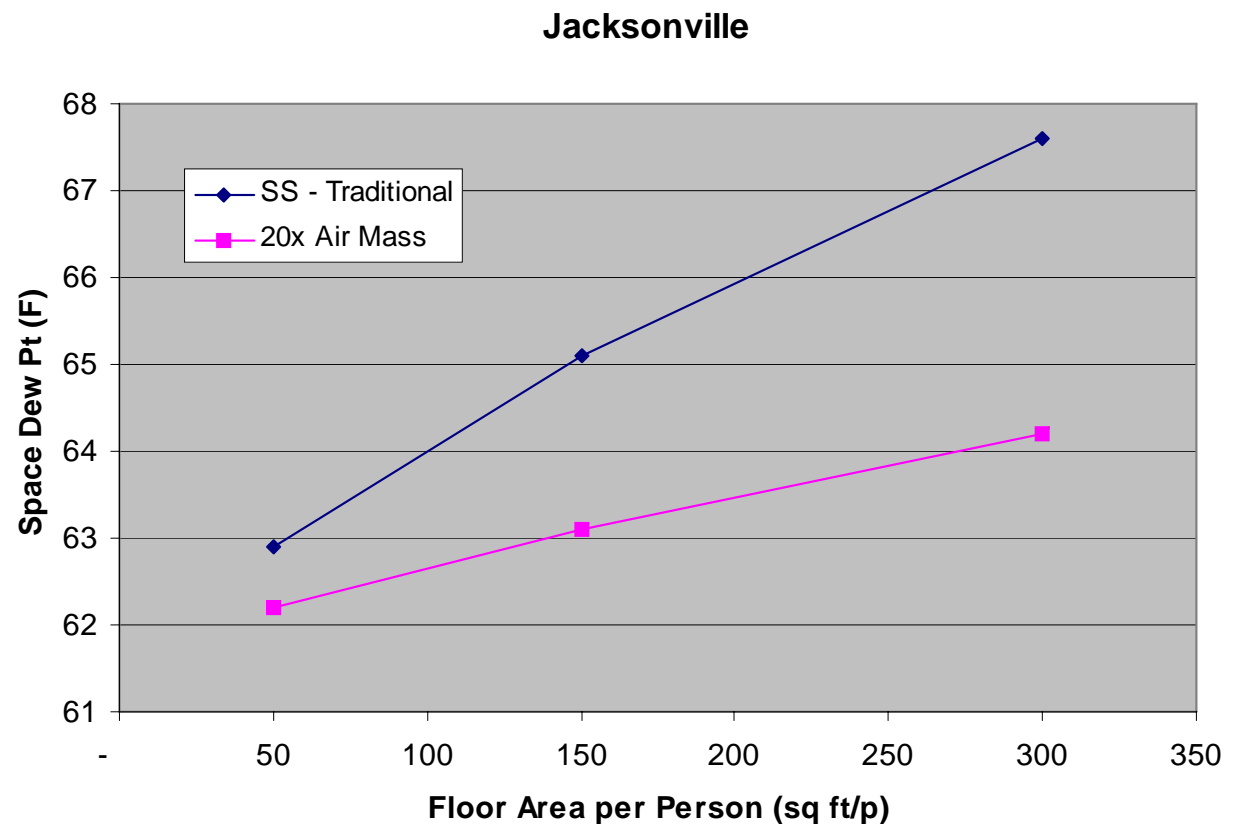
3.0°F – “error”

JCKSNVLL - July 18 - Supply Air TDP = 52F



# Moisture Capacitance Matters More in “Low Occupant Density” Applications

- The “Over-prediction” with Steady State Calculations can exceed 3°F

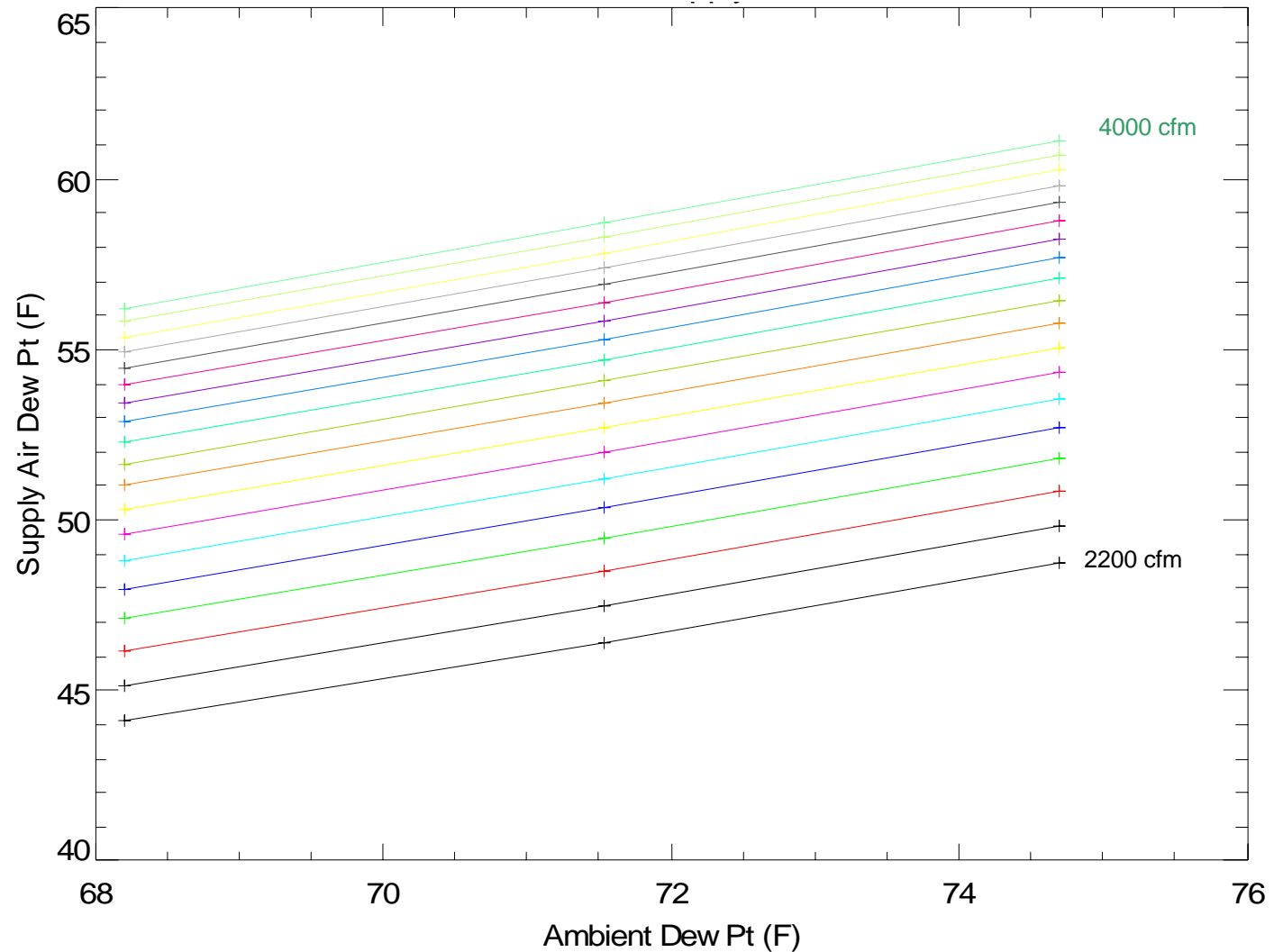


# Why Does This Matter?

- Steady-State moisture load calculations over-predict the necessary supply air dew pt
- It takes a “larger” MAU to provide lower supply air dew pt for a given airflow
- Everyone knows that you can't do simple steady state calculations for sensible cooling loads ...
- So why do we do it for moisture?

# MAU Catalog Data

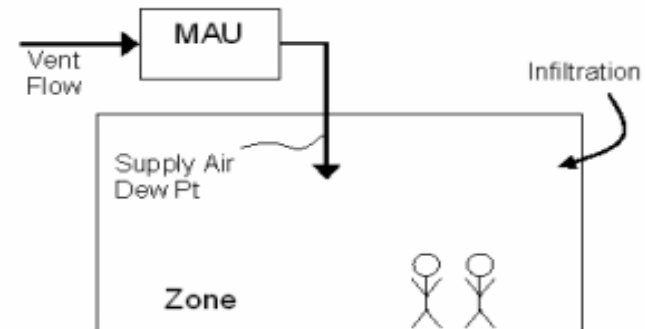
- Nominal 4000 cfm MAU (at 61F dpt)
- Only provides 2200 cfm at 49F dpt



# Web Tool Available

## Make-Up Air Unit Moisture Loads Tool

This tool demonstrates the effects of moisture capacitance on the latent sizing calculations for a makeup air unit (MAU). For peak humidity periods, the latent load in the space can be much larger than the dehumidification capacity but reasonable space conditions can still be maintained. The reason is moisture capacitance. The moisture capacitance is estimated by taking the space volume and multiplying by the moisture capacitance factor (typical values range from 1 to 20). By selecting different inputs below, the effects of moisture capacitance on space humidity levels can be demonstrated.



### Select Parameters

Background Infiltration Rate (Air Changes per Hour)	<input type="text"/>
Occupant Activity Level:	Seated at Rest (140 Btu/h Latent) <input type="button" value="v"/>
Number of Occupants	<input type="text"/>
Volume of Space (cubic feet)	<input type="text"/>
MAU Supply Air Dewpoint (deg F)	<input type="text"/>
MAU Supply Air Flow Rate (cfm)	<input type="text"/>
Select TMY2 Weather Data:	Albany, New York <input type="button" value="v"/>
Moisture Capacitance Factor	<input type="text"/> (Enter 0 for steady state calcs)

### Select Occupancy Schedule (Check boxes for hours of operation)

Pre-defined Schedules:	<input type="checkbox"/> 12:00am - 12:00am	<input type="checkbox"/> 6:00am - 6:00pm	<input type="checkbox"/> 8:00am - 5:00pm	<input type="checkbox"/> Match Ventilation
<input type="checkbox"/> 12:00 am - 1:00 am	<input type="checkbox"/> 1:00 am - 2:00 am	<input type="checkbox"/> 2:00 am - 3:00 am	<input type="checkbox"/> 3:00 am - 4:00 am	
<input type="checkbox"/> 4:00 am - 5:00 am	<input type="checkbox"/> 5:00 am - 6:00 am	<input type="checkbox"/> 6:00 am - 7:00 am	<input type="checkbox"/> 7:00 am - 8:00 am	
<input type="checkbox"/> 8:00 am - 9:00 am	<input type="checkbox"/> 9:00 am - 10:00 am	<input type="checkbox"/> 10:00 am - 11:00 am	<input type="checkbox"/> 11:00 am - 12:00 pm	



# Summary

- Traditional steady state moisture load calculations “over-predict” the required supply air dew point
  - predicted supply air dew point can be too low by 1-3°F
  - error is larger in low occupancy buildings
- Providing 1-3°F warmer supply air dew pt reduces MAU size and cost
- Web tool is available to illustrate the concept