

Implications of GHP Fan Control Strategies on Humidity Control in Commercial Applications

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Fan Control Affects Latent Performance

- How is ventilation air supplied?
 - Provided directly to heat pump inlet
 - Provided directly to space

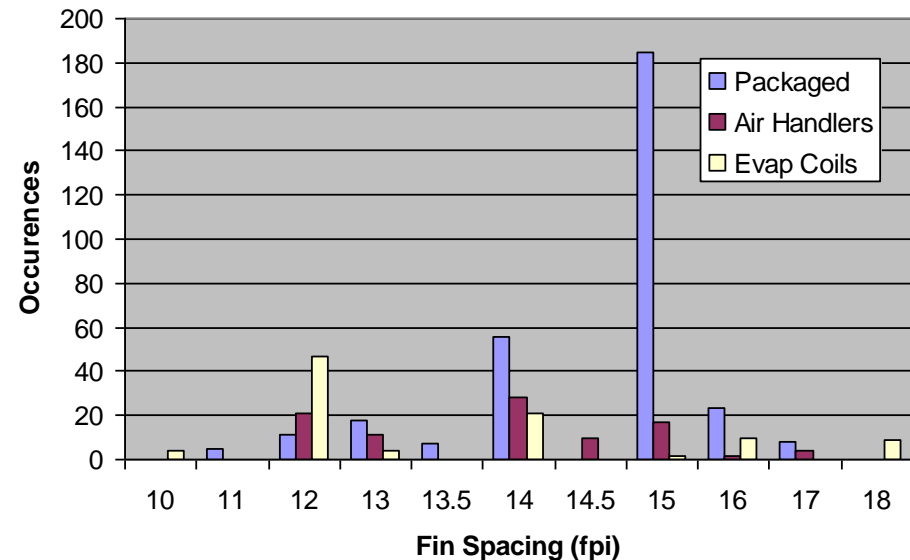
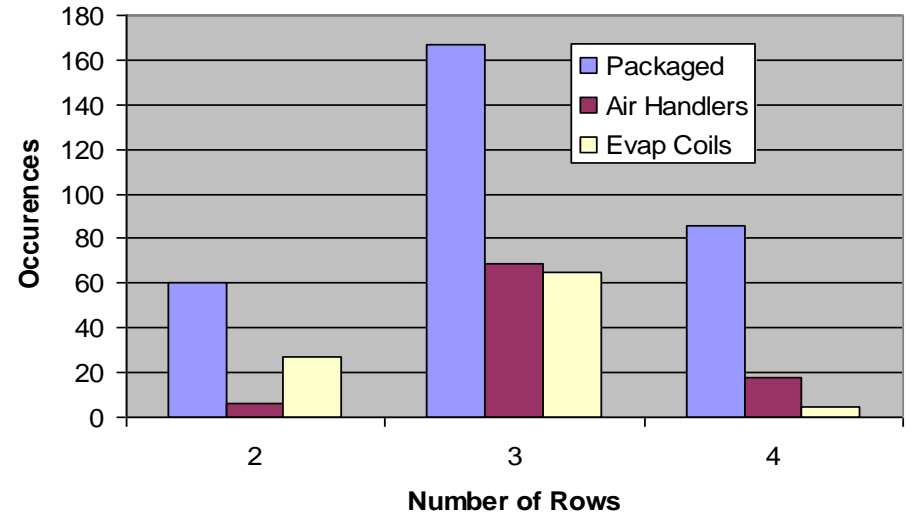
- If supply fan operates continuously, then latent capacity degrades at part load

Steady-State Cooling Coil Performance

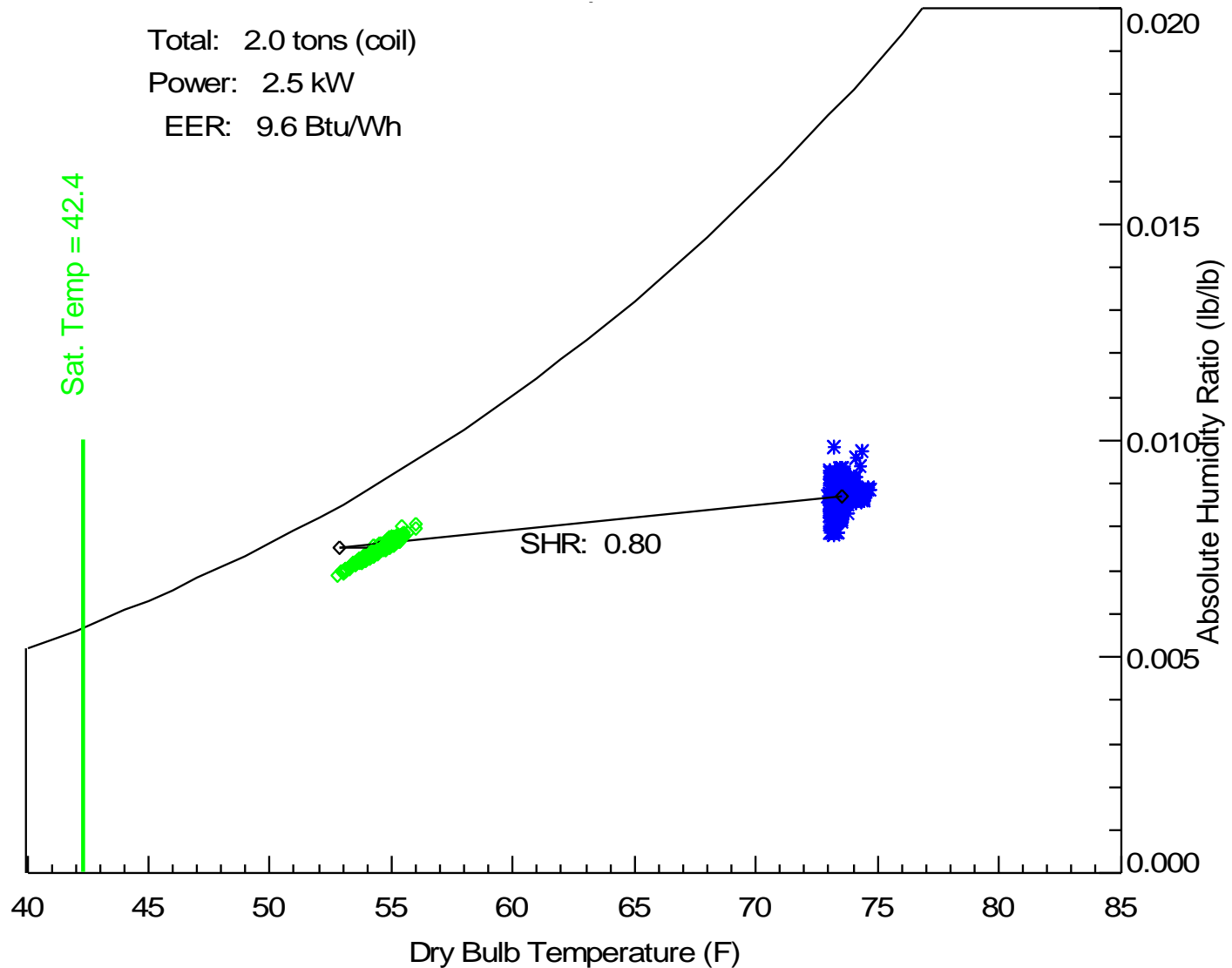
- AC coil performance is well understood at steady-state conditions
- GHP is similar to conventional AC
 - Coils provide both sensible cooling and moisture removal (latent cooling)
 - SHR: *sensible heat ratio*.....fraction of total cooling that is sensible
 - Colder coil surfaces remove more moisture
 - Reduced air flow provides more moisture removal (for DX coils)
 - Higher condenser temperatures increase SHR
 - i.e., high ground loop temperatures

Equipment Survey

- Reviewed specifications for 500 commercial and residential AC units
- Goal was to determine
 - range of common coil geometries
 - Variation by equipment type
- Typical DX AC coil is 3-rows, 1 fpi



Typical AC Coil Performance



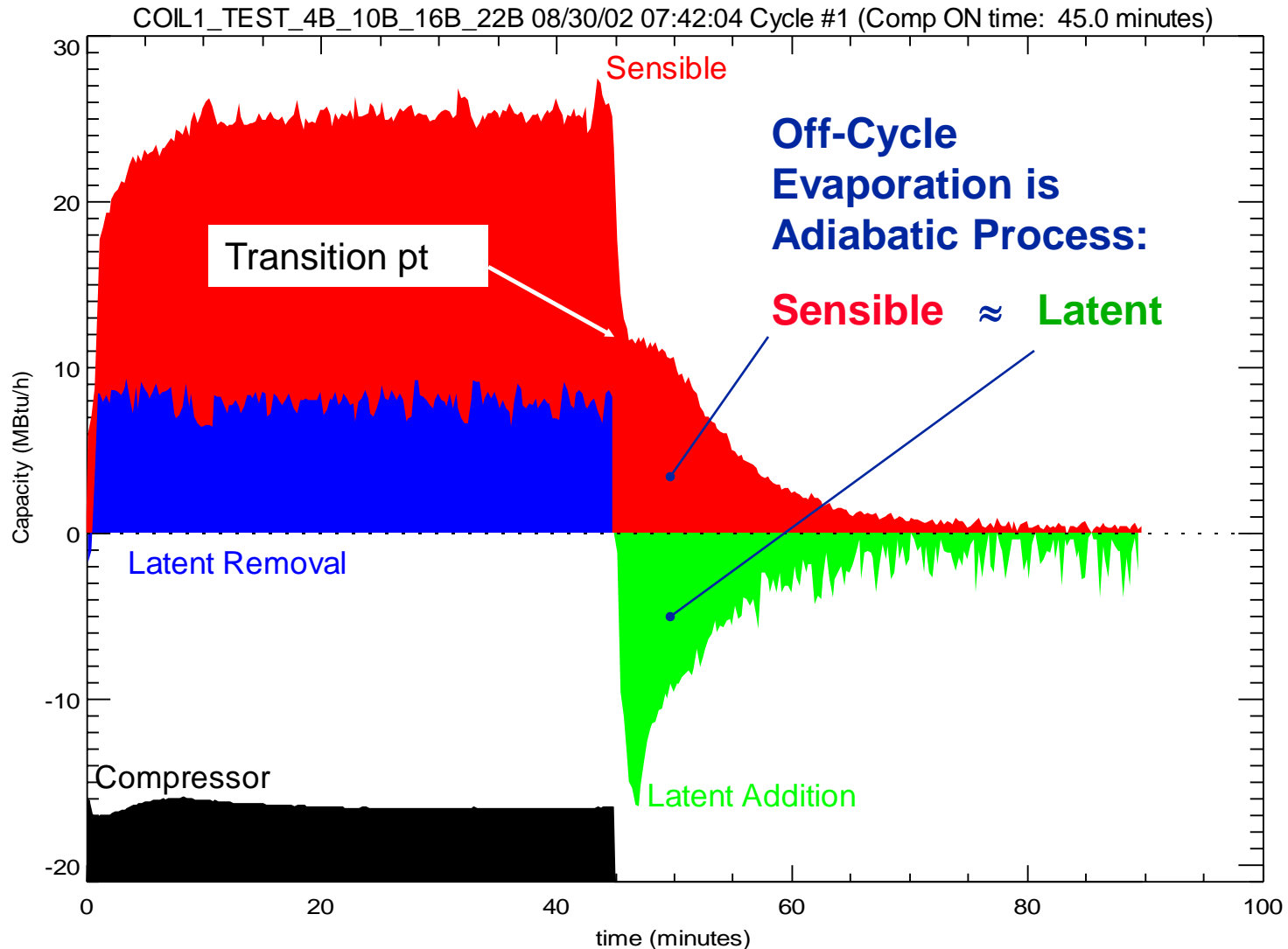
Part Load Performance

- GHP cycles compressor ON and OFF based on a space thermostat
- The portion of time the coil operates (i.e., the runtime fraction) is longer when cooling loads are greater

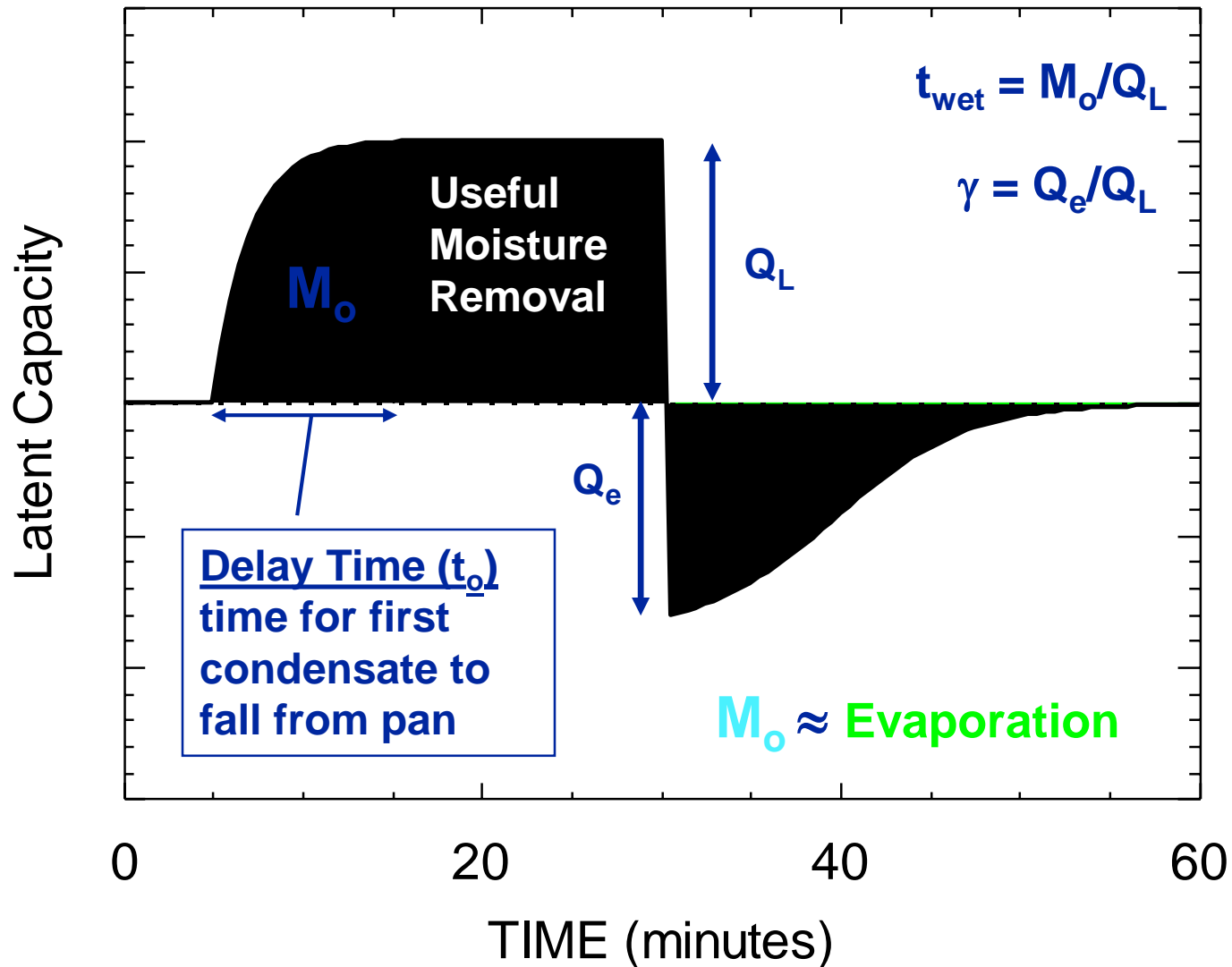
$$\text{RTF} = \frac{\text{ON}}{(\text{ON} + \text{OFF})}$$

- How do sensible and latent capacity vary under cyclic conditions?

Sensible and Latent Capacity With Continuous Supply Air Fan Operation



Latent Degradation Concepts

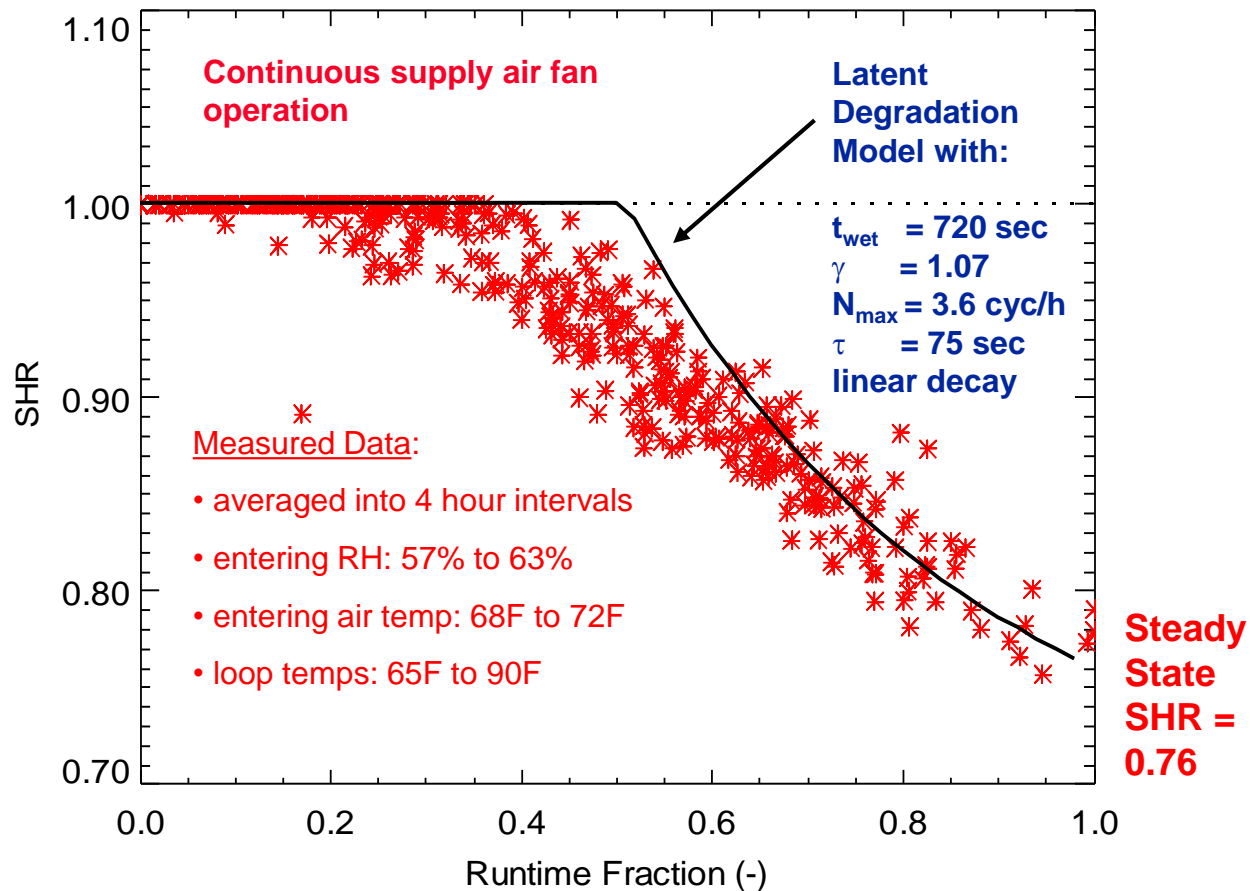


LHR Degradation Model

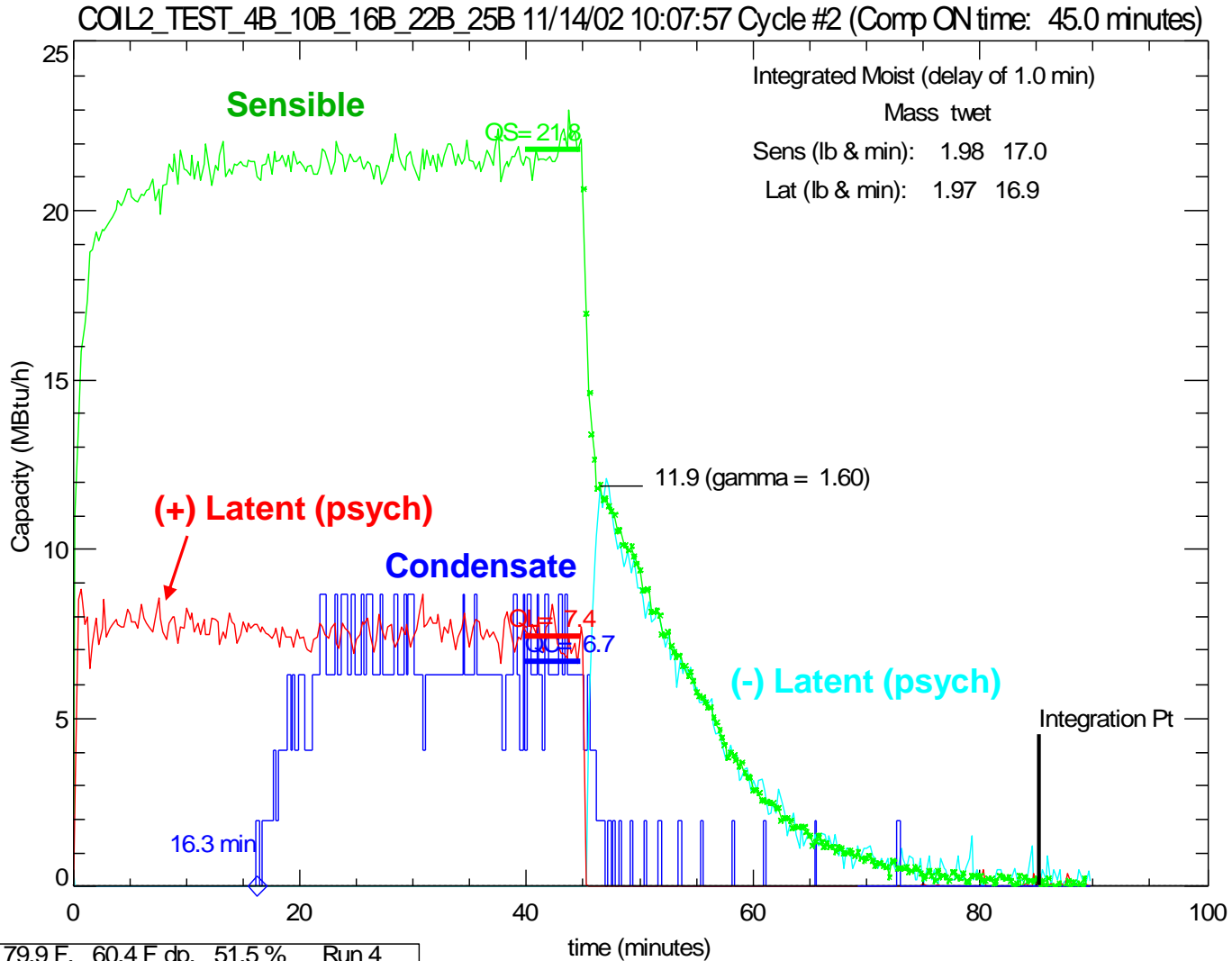
- Borrowed approaches used to develop part-load efficiency degradation function for SEER test procedure
- Used same part-load assumptions (C_d):
 - AC at startup described by a time constant
 - Cycling rate driven by thermostat curve
- Additional latent assumptions:
 - Coil surfaces hold a fixed amount of moisture (M_o)
 - Off-cycle coil is like an evaporative cooler

Comparing LHR Model to Field Tests

- Henderson (1998) compared the model results to field data collected on a 3-ton residential geothermal heat pump



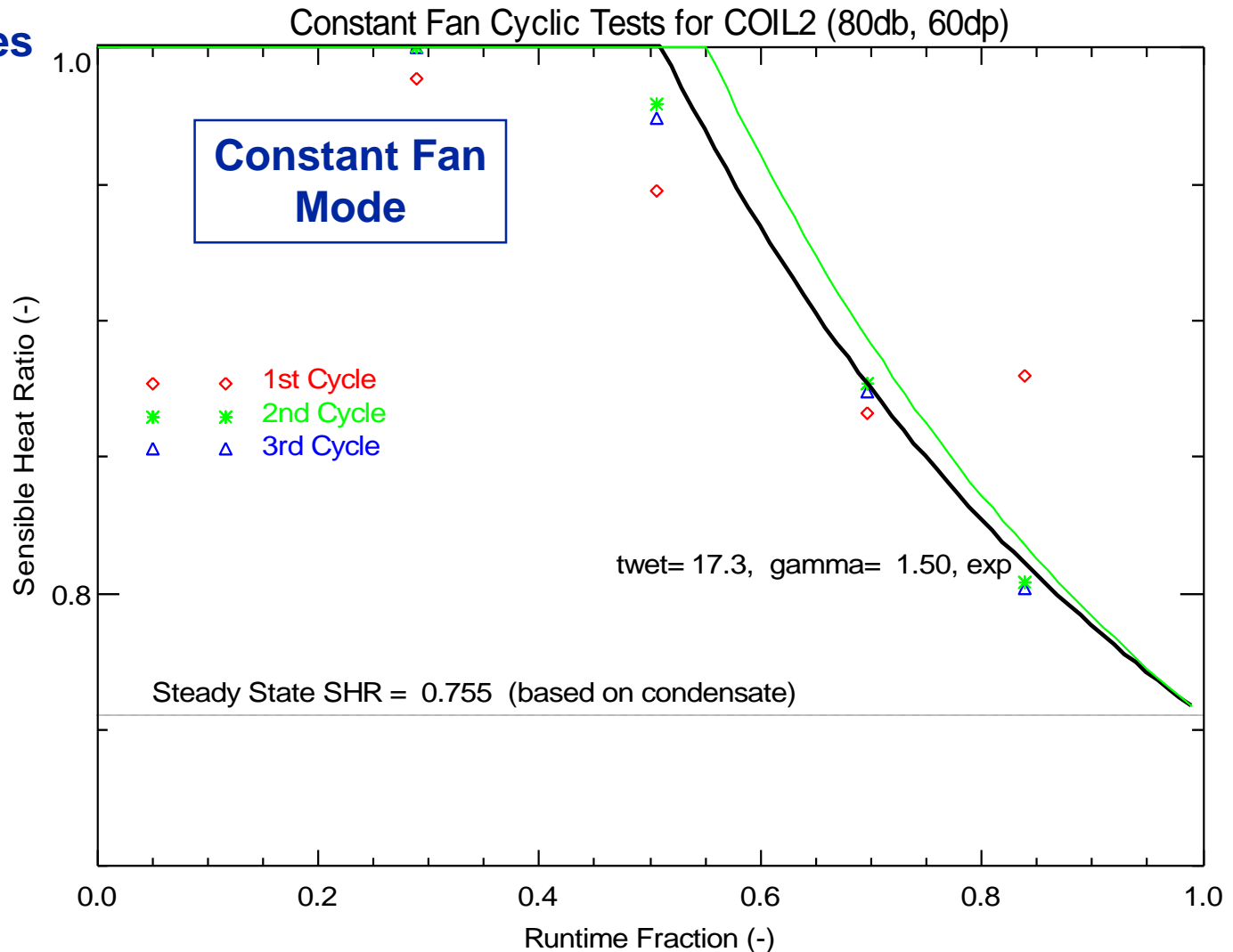
Actual Laboratory Results



79.9 F, 60.4 F dp, 51.5 % Run 4
1.5 hz, 76.6 psi, 967 cfm, 30.04 in Hg

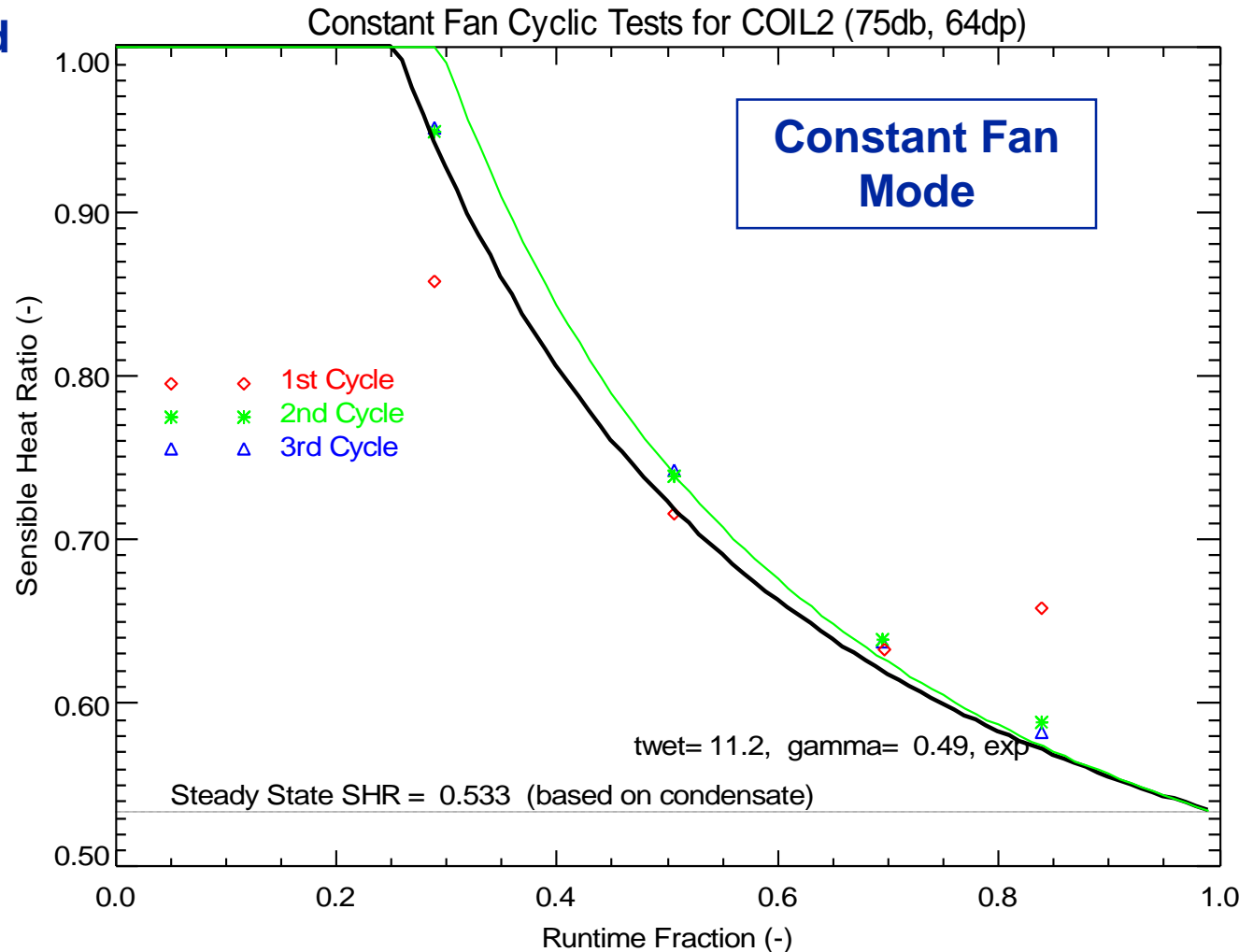
Laboratory Testing – SHR Degradation

**LHR Model matches
measured data at
nominal entering
conditions
(80F db, 60.4F dp)**



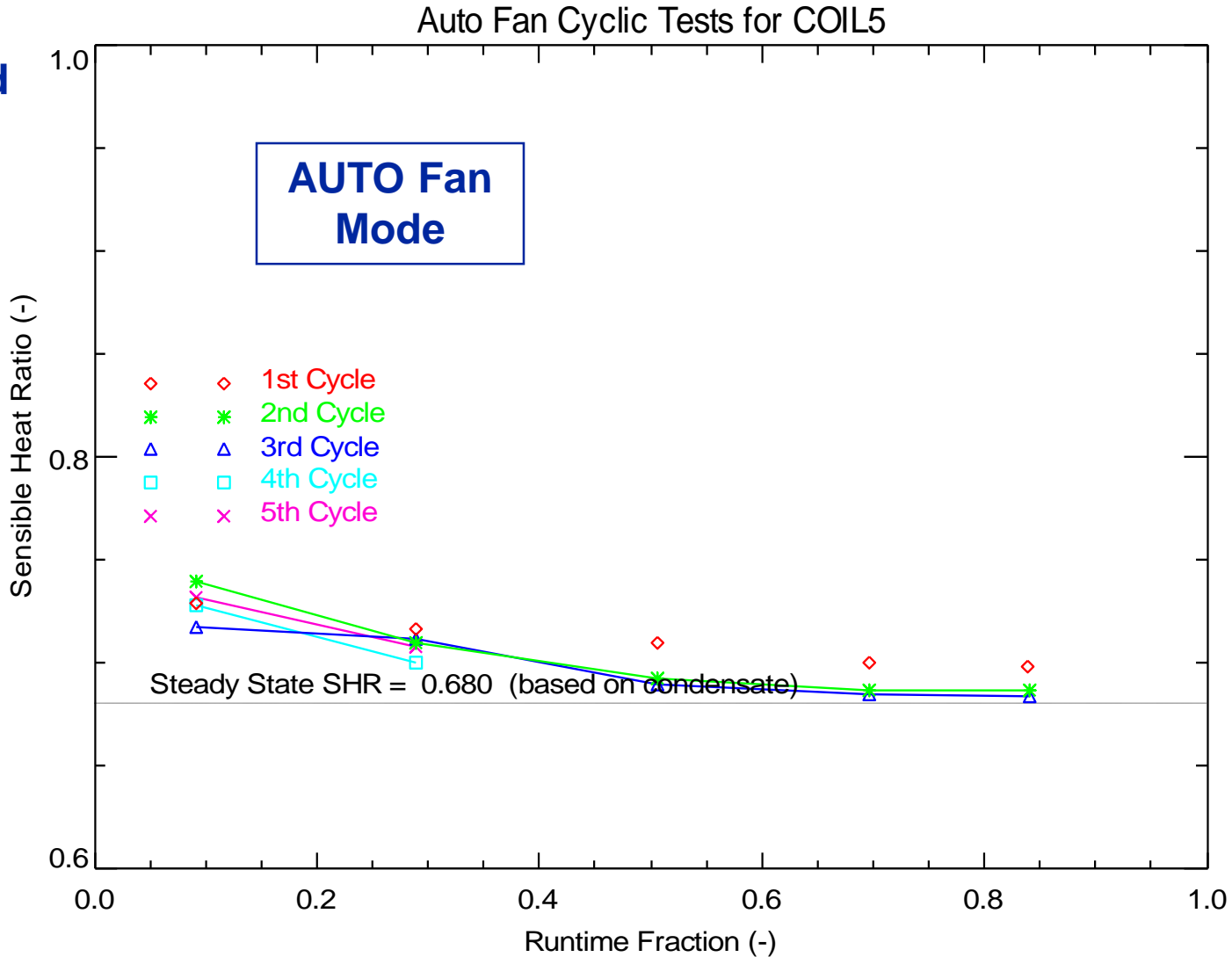
Laboratory Testing – SHR Degradation (cont.)

**Model also matched
measured data at
other entering
conditions!
(75F db, 64F dp)**



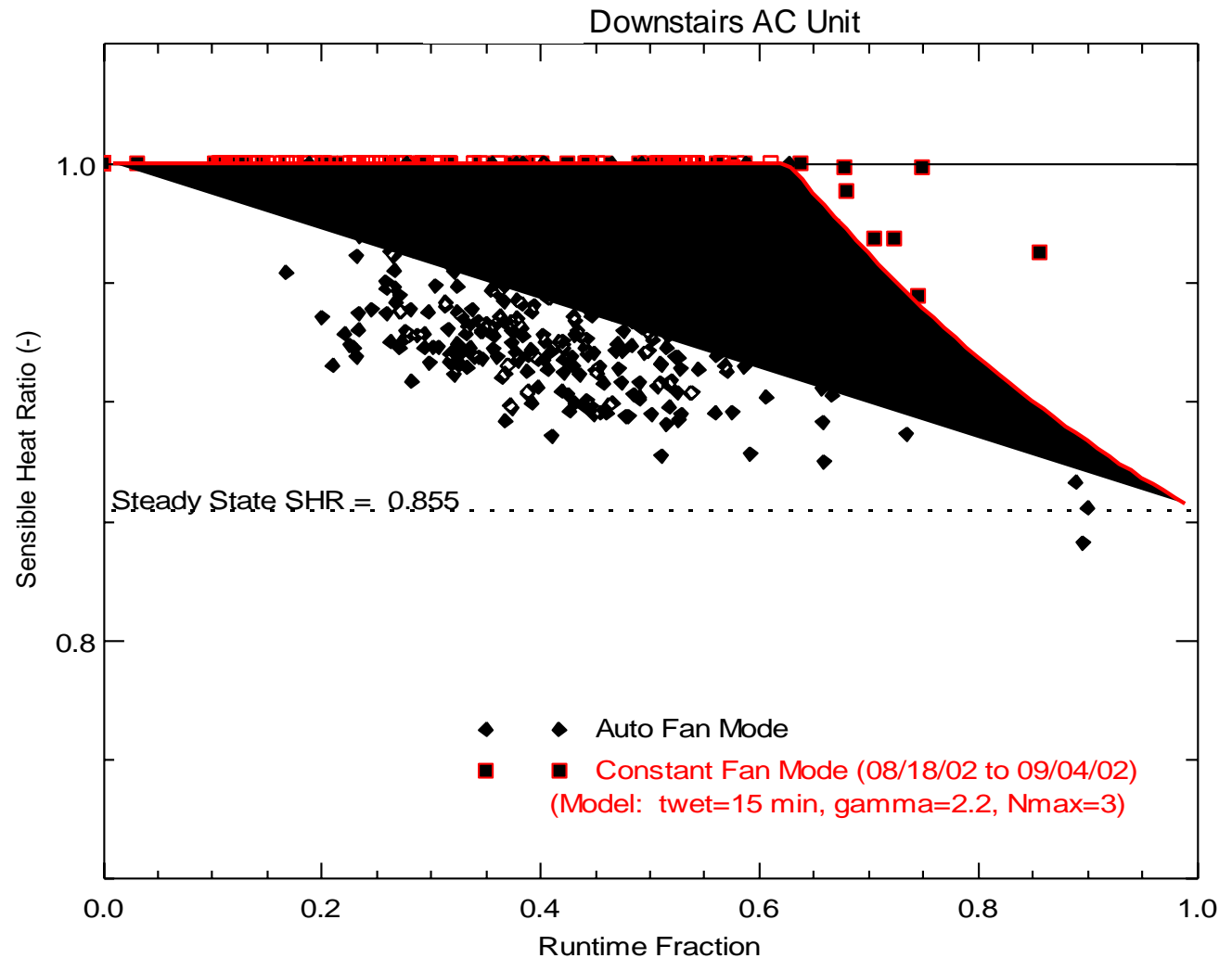
Laboratory Testing – SHR Degradation (cont.)

Coil 5 showed impact in AUTO Fan Mode!



Field Testing – SHR Degradation

Significant
AUTO fan
degradation!

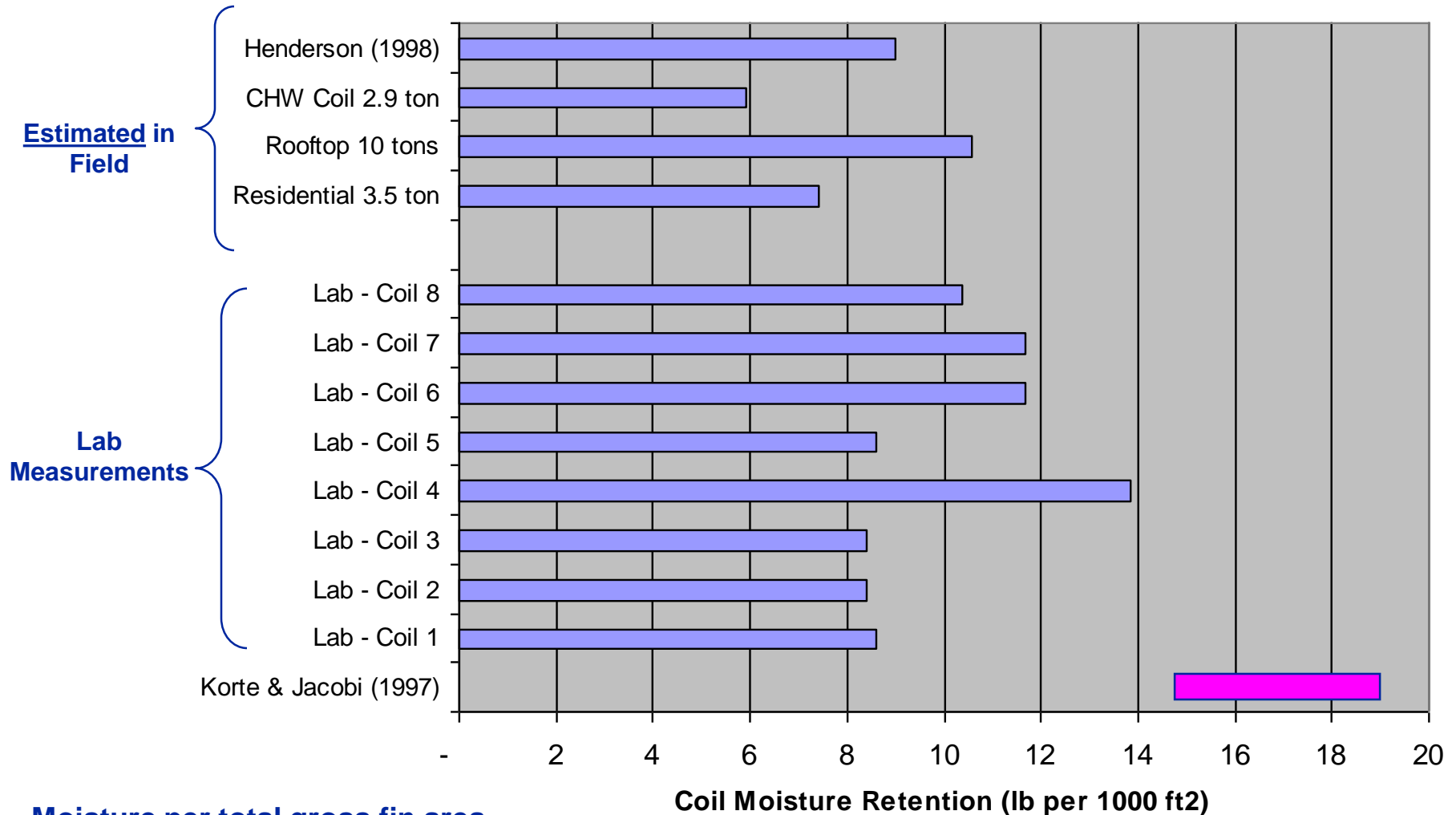


Measured Performance Parameters

	Size (tons)	Fin Surface Area (ft ²)	Retained Moisture Mass		Cond Delay Time (min)	t _{wet} (min)
			(lb)	(lb/kft ²)		
Coil 1 (Slanted slab, 3 row, 13 fpi, plain fins, orifice)	2.9	243.8	2.1	8.6	13.5	16.5
Coil 2 – Normal Flow (A-coil, 3 rows, 15.5 fpi, lanced sine-wave fins, TXV)	2.4	237.8	2.0	8.4	16.3	17.0
Coil 2 – Low Flow (A-coil, 3 rows, 15.5 fpi, lanced sine-wave fins, TXV)	1.4	237.8	2.0	8.4	32.5	29.0
Coil 4 (vert. slab, 2 rows, 14 fpi, wavy fins, orifice)	1.8	137.4	1.9	13.8	23.5	18.5
Coil 5 (slanted slab, 4 rows, 12 fpi, wavy fins, orifice)	2.3	162.7	1.4	8.6	11.5	9.5
Coil 6 (A-coil, 3 rows, 13 fpi, wavy fins, TXV)	1.7	231.1	2.7	11.7	34.0	33.
Coil 6 – High Flow (A-coil, 3 rows, 13 fpi, wavy fins, TXV)	2.0	231.1	2.7	11.7	27.0	27.
Coil 8 – Chilled Water (vert. slab, 4 rows, 10 fpi, wavy fins, 46°F CHW)	1.5	135.0	1.4	10.4	26.5	26.5

- Notes:
- 1- Cooling capacity includes sensible and latent cooling at nominal conditions. Nominal conditions correspond to ASHRAE Test A test point.
 - 2- Surface area is gross fin area (coil face area x coil depth x fin spacing x 2).
 - 3- Condensate delay time and t_{wet} are at nominal conditions.

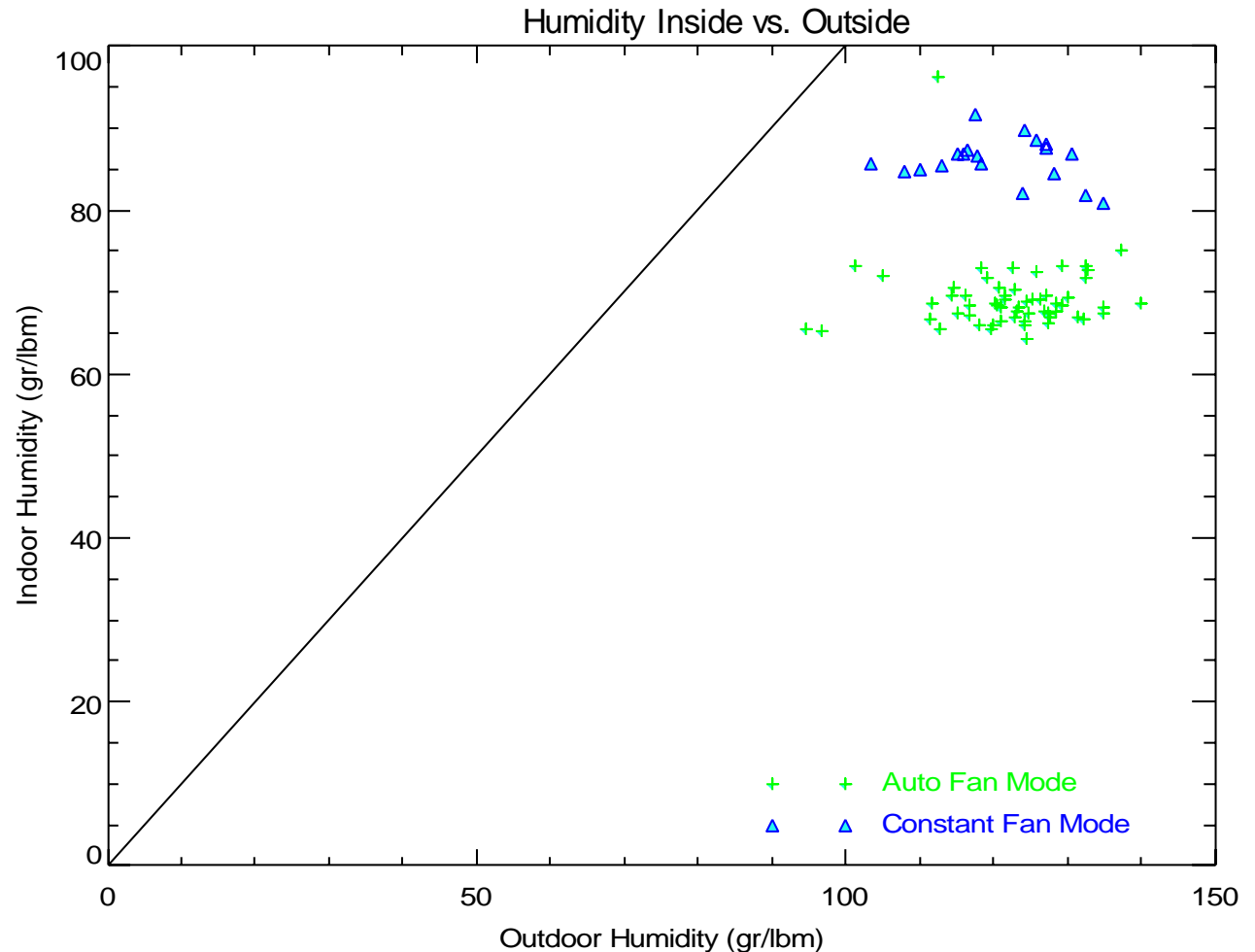
Lab + Field: Retained Moisture



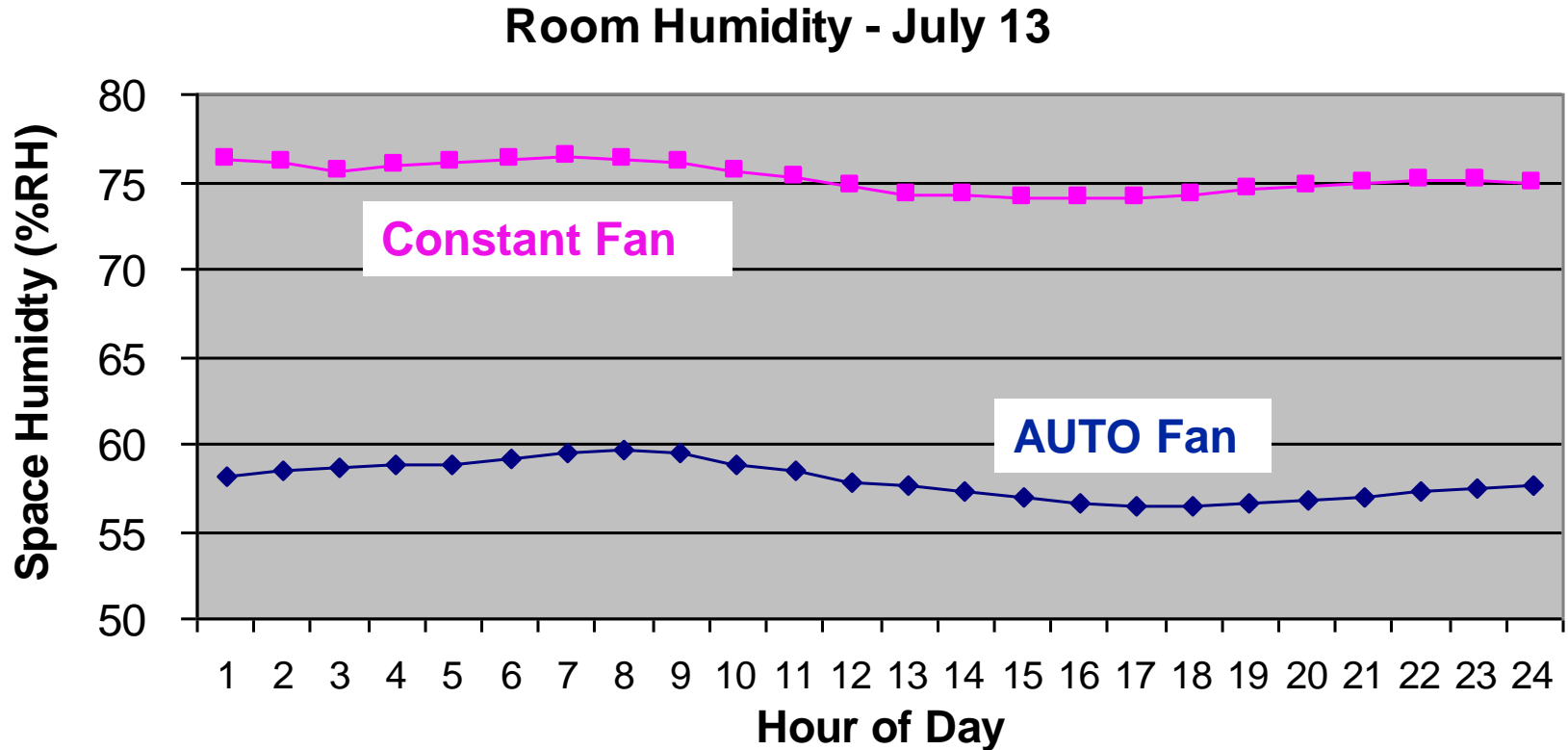
Moisture per total gross fin area
(gross area = face area x depth x fin spacing x 2)

Field Testing – Impact of Constant Fan

- Large penalty for constant fan operation
- Even for a dual-capacity AC unit



Impact on Space Humidity Levels



**TRNSYS Model of Hotel Room
in Hawaii**

What Control Strategies Help?

Do fan delay strategies help?

- ❑ No...not until about a 10 minute delay

Two-speed or staged capacity heat pumps?

- ❑ Yes...increases runtime fraction (depends on coil split & fan control)

Reduced airflow during off cycle?

- ❑ Yes...would minimize off cycle evaporation

Cycle the fan with the compressor?

- ❑ Yes...whenever possible

Summary

- ❑ GHPs are like Air Conditioners, only they're geothermal
 - ❑ Follow same steady state latent/sensible trends as conventional DX coil
 - ❑ Subject to the same part load effects
- ❑ With constant fan, latent removal disappears below half load
- ❑ To maintain latent capacity
 - ❑ Cycle fan with the compressor (when possible)
 - ❑ Slow the fan down during off cycle
 - ❑ Add more capacity stages